

# Hydraulically actuated clutches and spring-applied brakes

clutch/brake combined units



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Thanks to the large force available for actuation and the use of the steel/sinter combination, which has very low wear characteristics, hydraulically actuated Ortlinghaus Sinus® multi-plate clutches, brakes and clutch/brake combined units are suitable for a range of applications in mechanical engineering, motor vehicles and transmission systems.

The cooling oil, which is led through the plates in a precise manner, allows the heat, which varies depending on the particular application, to be dissipated efficiently.

To a very large extent these clutches and brakes are wear free and require no maintenance!

## Clutch/brake combined units for wetrunning

### 1/2

#### **Series 0023 and 0123**

Clutch/brake combined units are recognised as being safe and reliable for driving presses, metal-forming and machine tools, shears and similar machines. They conform with the well known safety requirements as laid down in the relevant EC directives.

The torque transmitting capability and the switching capacity can be influenced over a wide range, by the correct selection of the number of friction surfaces.

The heat generated from a particular application can be dissipated efficiently with the aid of a carefully designed cooling oil system.

The shaft hub connection is with a double keyway in the case of series 0023; in the case of series 0123, either a double keyway or a locking assemble can be selected.

Friction combinations: Steel/sinter linings Operating pressure: 60 bar.

Pressure and cooling oil intake: Via rotary inlet through the shaft.

Application: In stamping, embossing and drawing presses, in automobile body presses, shears and similar machines.

## Sinus® multi-plate clutches for wet-running, standard version

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## **Series 0021-007**

This version can be put to use universally as a compact machine transmission clutch. Friction combination: Steel/sinter lining Operation pressure: 18 to 20 bar Pressure and cooling oil intake: Via rotary inlet through the shaft.

Examples of application: In transmissions for machines and vehicles, stationary and mobile cranes, other lifting devices.



## Sinus® multi-plate clutches for wet-running; version for high torques

4 Series 0021-3.3

A clutch for heavy-duty drives; normal version with shoulder housing and without "emergency engagement facility". A version with "emergency engagement facility" is also available.

Large hub bores are possible. These clutches can be manufactured to conform to the acceptance conditions of the classification institutions for marine engineering.

Friction combination: Steel/sinter lining Pressure and cooling oil intake: Via rotary inlet through the shaft.

Operating pressure: 25 bar

Areas of application: Used in marine engineering for reversing gears, variable pitch propeller drives and multi-motor drives.

## Sinus® multi-plate clutches for wet-running; version for high thermal loading

5 Series 0-002

A clutch for heavy duty drives of all types in which a high level of engaging/disengaging work has to be performed; normally supplied with a flange or shoulder housing and without "emergency engagement facility". A version with "emergency engagement facility" is also available. These clutches can be manufactured to conform to the acceptance conditions of the classification institutions for marine engineering.

Friction combination: Steel/sinter lining Pressure and cooling oil intake: Via rotary inlet through the shaft

Operating pressure: 24 bar

Areas of application: Used in marine engineering for reversing gears, variable pitch propeller drives, multi-motor drives and power take off drives.

## Spring-applied, hydraulically released multi-plate brakes for wet- or dry-running, non centering

6 Series 0022-... 0/-... 9

Spring-applied fail safe brakes with a housing which does not have a centering function; for this reason these brakes are primarily used on shaft ends and outside gearboxes.

Variations include a closed end flange and an open end flange, which would allow a shaft to pass through.

The "wet-running plates" are lubricated initially by soaking in oil or by means of oil stored in the

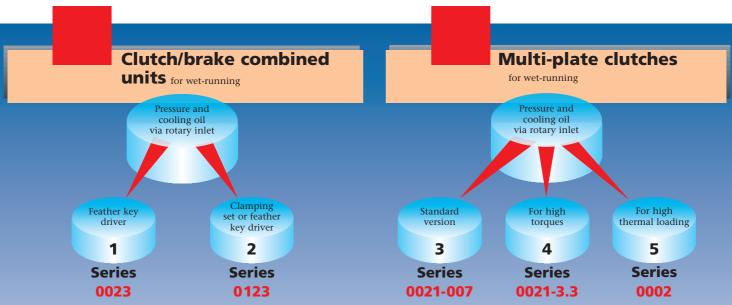


plate chamber, depending upon the particular application.

The brakes can be released mechanically in the case of a hydraulic system failure (emergency release facility).

Friction combination: Steel/sinter lining for wet or dry-running

Pressure oil intake: On outside of diameter of stationary cylinder.

Operating pressure: max. 320 bar

Application: Extensively used throughout the field of mechanical engineering especially as brakes for hydraulic motors used in cranes and winches.

## Spring-applied, hydraulically released multi-plate brakes for wet- and dry-running with internal and external centering facility

### 7 Series 0022-. . 1

Spring-applied fail safe brake in which the housing has both a centering and a load carrying function.

The "wet-running plates" are lubricated initially by soaking in oil or by means of oil in the chamber, depending upon the particular application. The brakes can be released mechanically in the case of a hydraulic system failure (emergency release facility).

Friction combination: Steel/sinter lining for wet or dry-running

Pressure oil intake: On outside diameter of stationary cylinder.

Operating pressure: max. 320 bar

Application: Generally within a transmission between motor (hydraulic motor) and driven device.

## Spring-applied, hydraulically released multi-plate brakes for wet- and dry-running with dual internal centering facility 8 Series 0022-. 20

This is a short brake for use in designs where space is limited. The housing has both a centering and a load carrying function.

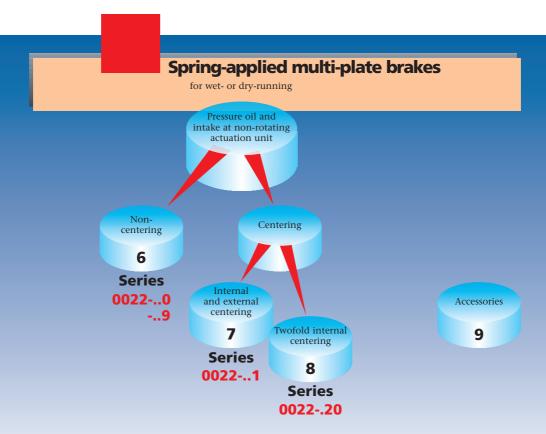
Friction combination: Steel/sinter lining for wet or dry-running

Pressure oil intake: On outside diameter of stationary cylinder

Operating pressure: max 320 bar

Application: In gearboxes of every type, cranes

and other lifting devices.





## 9 Accessories

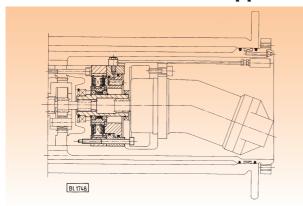
Ortlinghaus has available a comprehensive range of accessories for operating hydraulically actuated clutches and brakes and to permit these to be correctly intergrated into the particular machine design.

We can supply!

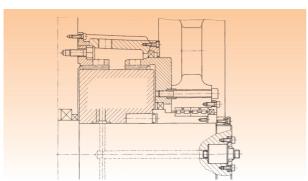
- Single and multi-channel rotary inlets (see also special catalogue)
- Press safety valves,

- Complete clutch-brake controls in modular design
- Complete hydraulic power packs for supplying clutch/brake combined units with pressure and cooling oil
- Cooling systems in the form of oil/air or oil/water heat exchangers
- Sealed housing covers for enclosing clutch/brake combined units.

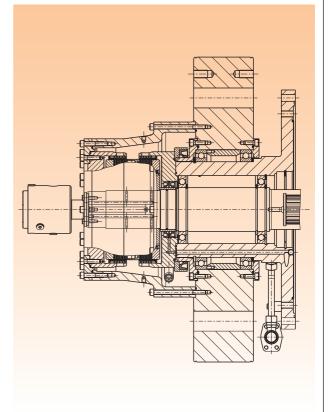
## **Application examples**



Spring-applied, hydraulically released multi-plate brake, series **0022-304** fitted in the drive of a winch.



Spring-applied, hydraulically released multi-plate brake, series **0022-601** fitted in the travel gearbox of a caterpillar-type excavator.



Hydraulically actuated clutch/brake combined unit, series **0123**, fitted in a press drive.

No.	Series	Torque range	Hub bore	<b>External diameter</b>
		Nm	mm	mm
1	0023 Clutch	2500 to 960000	45 to 375	230 to 1040
	Brake	500 to 240000		
2	0123 Clutch	12000 to 99000	70 to 200	380 to 560
	Brake	4800 to 28800		
3	0021-007	200 to 4000	18 to 82	95 to 252
4	0021-3.3	11200 to 630000	50 to 400	280 to 1000
5	0002	9000 to 300000	50 to 260	315 to 750
6	00220-/9	33 to 120000	18 to 350	83 to 910
7	00221	70 to 5900	20 to 110	135 to 315
8	002220	50 to 6100		120 to 345

## Fax questionnaire for the designing of plates

Please complete in block capitals!



Sender:	Recipient:		
	Ortlinghaus-Werke GmbH		
Name, first name	Kenkhauser Straße 125 · Postbox 1440		
,	42907 Wermelskirchen · Germany Tel. +49 2196 85-0 · Fax +49 2196 855-444		
Company	info@ortlinghaus.com · www.ortlinghaus.com		
1 /			
Department Telephone (extension)	for the attention of (if known)		
	Fax-No. +49 2196 855-444		
Fax			
For clutches and brakes:	For press clutches and brakes:		
Actuation type:	Actuation type: pneumatic hydraulic hydraulic		
hydraulic pneumati	c Arrangement: Clutch and brake separated Auxiliary brake		
spring-applied 🔲	Clutch and brake separated Auxiliary brake Clutch and brake combined with auxiliary brake		
Drive machine:	Fitting position: Rotary axle horizontal  Vertical		
Elektric motor Combustion engin	Shaft diameter d = mm		
Hydraulic motor U Other:			
Drive situation:	Driver on the shaft:		
Fisher and an address of the second s	Feather keys Clamping set/contraction disk		
<b>Fitting situation</b> :  Rotary axis horizontal  Vertica	Actuation anaccura air process D - bar		
Rotary axis horizontal $\square$ Vertica Exposed $\square$ In closed housing	II		
_	Oil pressure		
± -	$P_{\rm p} = $ bar		
on drive output $d_2 = $ $P$ Motor data: Capacity $P = $ $P$	$P_{max} = 0$		
Motor data: Capacity $P = $	Series no.		
Torques on clutch or brake:	<b>Version characteristics</b> (e.g. mode of securing plates)		
Switchable torque $M_s =$	Nm		
Transmittable torque $M_{\ddot{u}} = $	Machine type:		
Load torque $M_L =$	Working mode: single stroke continuous run		
Course of M <sub>L</sub> , when this changes:	<b>Motor capacity:</b> P = kW, at n = rpm		
E/	<b>Max. pressing/shearing force</b> F = kN		
Initial input drive speed: $n_{10} = $	Working angle at BDC $\alpha =$ deg.		
Initial output drive speed: $n_{20} = \phantom{aaaaaaaaaaaaaaaaaaaaaaaaaaaaaaaaaaa$	min-1 working neight at BDC n = mm		
Max. relative speed ratio: <sup>3</sup> <sub>n</sub> =n	Eccentic radius $r = \underline{\hspace{1cm}} mm$		
Conditions at switching:	Lenght of the connecting rod 1 = mm		
Stationary Full load Without loa	Eccentric speed $n_E = \underline{\qquad} \min^{-1}$		
Switching frequency $S_h = $ h	Clutching speed $n_K = \underline{\hspace{1cm}} min^{-1}$ (state switching speed without fail)		
Acceleration/deceleration time $t_3 = $ s			
Moment of inertia about clutch	Moment of inertia of all		
or brake shaft axis:	masses to be braked $J = kgm^2$		
Input drive side $J_A = $ l			
Output drive side $J_L = $ 1	kgm <sup>2</sup> Course of J, if this changes		
Course of $J_A \square$ , $J_L \square$ ,	Ram mass including tool m = kg		
when these change:	3		
Further details:	<b>Load torque at braking</b> $M_L$ =Nm		
	Course of M <sub>L</sub> if this changes		
	<b>Desired braking angle</b> $\varsigma = $ deg.		
	<b>Desired braking time</b> $t_{Br} = \underline{\hspace{1cm}} s$		
	Envisaged solenoid valve		
	Flywheel external diameter D <sub>c</sub> = mm		